TECHNICAL BULLETIN

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## \#1 How To Determine Gear Pump Flow And Displacement

Today we will be covering how to determine gear pump flow, and displacement.
Let's first cover how a gear pump works. The oil enters the gear pump from the (Inlet) tank and travels around the outside of the gear and forced out of the pressure port. See IMG. 1
IMG. 1


There are 2 each brass side pressure plates that are $1 / 4^{\prime \prime}$ thick on both sides of the gear. These keep the oil retained in the teeth from the side. See IMG.2,3

IMG. 2
IMG. 3


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When you measure the pump section you will have to subtract $1 / 2^{\prime \prime}$ because of the two $1 / 4^{\prime \prime}$ side pressure plates. Yellow arrow. See IMG. 4


IMG. 4

Looking at IMG.4, if the yellow arrow measures 2" the gear section is $1 \frac{1}{2 \prime \prime}$. See IMG. 5

IMG. 5


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To figure out what gear section you have inside the pump you will need to know the model of pump that is being used.

We generally use a P465 model pump for our 4500 P.S.I. hydraulic circuits. We are accustomed to using the pump code as the call out. See IMG. 6 Chart. If you measure a P465 pump at a $2^{\prime \prime}$ outside section like in IMG.4, it would be a code (15) and have a displacement of 5.4 cubic inches. See IMG. 6 Chart. (The cubic inches is how much oil the pump will put out with 1 rotation.)

We can now measure all the drive pump sizes and combine them to figure out what the displacement is for the drive circuit. As an example, a 765 P/U paired with a 200-6 vibro has 4 each (15)'s and 1 each (7) size drive pumps that are the P465 model.
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We know that a (15) is 5.4 cubic inches and a (7) is 2.7 cubic inches by using IMG. 6 Chart
See the below math on how to add the pumps together.

## $5.4 \times 4=21.6+2.7=24.3$ cubic inches total.

We now now that we have 24.3 cubic inches' total volume for our drive pumps.
The pump drive on the $765 \mathrm{P} / \mathrm{U}$ is a 1:1 gear ratio. The max engine R.P.M. is 2050.
We can know figure out the GPM that the P/U puts out. Using the IMG. 6 Chart formula for Pump Output (GPM).

See IMG. 6 Orange Arrow.

See the math formula below for the $765 \mathrm{P} / \mathrm{U}$.
Pump Output (GPM)

```
2050 X 24.3 = 215.65GPM
    231
```

Above we can see that a 765 P/U running at 2050 R.P.M. with a standard configuration of drive pumps will produce 215.65 GPM.

We will be addressing how pump efficiency comes into play with this formula in an upcoming technical bulletin.

IMG. 6 Chart on next page.

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|  |  |  | DISPLACEMENT (d) PUMP/MOTOR |  |  |  |  |  |  |  |  | (c) 1898. Commarelal intortoch Corp. |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| PUNP MODEL <br> SERIES |  | $\begin{aligned} & 16 \mathrm{H} \\ & 15 \mathrm{X} \end{aligned}$ | 30 31 | $50 / 51$ 258 | 37X | 75 76 | 125 | P315 | P330 | P350 | P385 | P415 | P430 | P450 | P485. |
| DISPLACEMENT (d) PBR IN. OP GBAR TIDTH |  | $\begin{gathered} 1.37 \\ \text { in } 3 / \text { Rev } \end{gathered}$ | $\begin{gathered} \left.\begin{array}{c} 1.87 \\ \ln ^{9} / \text { Rav } \end{array} \right\rvert\, \end{gathered}$ | $\begin{gathered} 2.55 \\ \text { in }^{3} / \text { Rev } \end{gathered}$ | $\begin{gathered} 3.00 \\ \operatorname{tin}^{3} / \text { Rer } \end{gathered}$ | $\begin{gathered} 4.10 \\ \ln ^{3} / \mathrm{Rsv} \end{gathered}$ | $\begin{gathered} 6.47 \\ \ln ^{3} / \mathrm{Ber} \end{gathered}$ | $\begin{array}{l\|} 1.24 \\ \mathrm{in}^{2} / \mathrm{Rev} \\ \hline \end{array}$ | $\begin{gathered} 1.97 \\ \text { in }^{3} / \mathrm{Rer} \\ \hline \end{gathered}$ | $\begin{gathered} 2.55 \\ \ln ^{2} / R \mathrm{ev} \\ \hline \end{gathered}$ | $\begin{gathered} 3.60 \\ \sin 3 / \mathrm{Rev} \end{gathered}$ | $\ln ^{1.24}$ | $\begin{gathered} 1.97 \\ 13^{3} / \text { Rer } \\ \hline \end{gathered}$ | $\begin{gathered} 2.55 \\ \mathrm{In}^{3} / \text { Rev } \\ \hline \end{gathered}$ | $\begin{array}{r} 3.60 \\ \sin 3 / \mathrm{Rev} \\ \hline \end{array}$ |
| MAXMMURECOMMBNDEDPRESSURB |  | $\begin{gathered} 2000- \\ 2500 \\ \text { PSI } \\ \hline \end{gathered}$ | $\begin{aligned} & 2500- \\ & 3000 \\ & \text { PSI } \\ & \hline \end{aligned}$ | $\begin{gathered} 2000- \\ 3000 \\ \text { PSI } \\ \hline \end{gathered}$ | 2000 PSI | $\begin{gathered} 2500- \\ 3000 \\ \text { PSI } \end{gathered}$ | $\begin{array}{r} 2500 \\ \text { PSI } \\ \hline \end{array}$ | $\begin{gathered} 3000- \\ 3500 \\ \text { PSI } \\ \hline \end{gathered}$ | $\begin{gathered} 3000- \\ 3500 \\ \text { PSI } \\ \hline \end{gathered}$ | $\begin{gathered} 3000- \\ 3500 \\ \text { PSI } \\ \hline \end{gathered}$ | $\begin{aligned} & 3000- \\ & 3500 \\ & \text { PSI } \\ & \hline \end{aligned}$ | $\begin{aligned} & 4000- \\ & 4500 \\ & \text { PSI } \\ & \hline \end{aligned}$ | $\begin{aligned} & 4000- \\ & 4500 \\ & \text { PSI } \\ & \hline \end{aligned}$ | $\begin{aligned} & 4000- \\ & 4500 \\ & \text { PSI } \\ & \hline \end{aligned}$ | $\begin{aligned} & 4000- \\ & 4500 \\ & \text { PS1 } \end{aligned}$ |
| GEAR SIZE | CODE | d | d | ¢ | d | d | d. | $8547$ | d 4 | d | d | d, | d. | d | d |
| 1/2 | (05) | . 685 | . 985 | 1.275 | 1.500 | 2.050 | 3.230 | . 620 | . 985 | 1.275 | 1.800 | . 620 | . 985 | 1.275 | 1.800 |
| 3/4 | (07) | 1.027 | 1.470 | 1.912 | 2.250 | 3.070 | 4.850 | . 830 | 1.470 | 1.912 | 2.700 | . 930 | 1.470 | 1.912 | 2.700 |
| 1 | (10) | 1.370 | 1.970 | 2.550 | 3.000 | 4.100 | 6.470 | 1.240 | 1.970 | 2.550 | 3.600 | 1.240 | 1.970 | 2.550 | 3.600 |
| 1-1/4 | (12) | 1.712 | 2.460 | 3.187 | 3.750 | 5.120 | 8.080 | 1.550 | 2.460 | 3.187 | 4.500 |  | 2.460 | 3.187 | 4.500 |
| 1-1/2 | (15) | 2.050 | 2.950 | 3.820 | 4.500 | 6.150 | 9.700 | 1.860 | 2.950 | 3.820 | 5.400 |  |  |  | 5.400 |
| $1-3 / 4$ | (17) | 2.397 | 3.440 | 4.460 | 5.250 | 7.1701 | 11.320 | 2.170 | 3.440 | 4.460 | 6.300 | 8 | 1.9 |  |  |
| 2 | (20) | 2.740 | 3.940 | 5.100 | 6.000 | 8.2001 | 12.940 | 2.480 | 3.940 | 5.100 | 7.200 |  |  |  |  |
| 2-1/4 | (22) |  |  | 5.730 | 6.750 | 9.2201 | 14.550 |  |  | 5.730 | 8.100 |  |  |  |  |
| $2-1 / 2$ | (25) |  |  | 6.375 | 7.500 | 10.2501 | 16.170 |  |  | 6.375 | 9.000 |  |  |  |  |
| $2-3 / 4$ | (27) |  |  |  |  | 11.275 |  |  |  |  |  |  |  |  |  |
| 3 | (30) |  |  |  | 9.000 | 12.30 |  |  |  |  |  |  |  |  |  |
| GENERAL FORMULAS |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| MOTOR SPEED (R.P.M.)$\text { Speed }=\frac{231 \times G P M}{d}=\text { RPM }$ |  |  | MOTOR TORQUE IN INCH POUNDS$\frac{\text { PSI } \mathrm{zd} \mathrm{d} \text { Efficiency }}{2 \pi}=\mathrm{T}$ |  |  |  |  | $\begin{aligned} & \text { PUXP OUTPUT (GPM) } \\ & \frac{\text { RPM } \mathrm{x} \text { d }}{231}=\text { GPY } \end{aligned}$ |  |  |  | PUMP INPUT (HP reqd.)$\frac{\text { GPM } £ \mathrm{PSI}}{1714 \geq \mathrm{Ef} \mathrm{~L}_{0}}=\mathrm{HP}$ |  |  |  |
| HORSEPOMER$\frac{\text { TORQUE (in.Ibs.) I RPM }}{63025}=H P$ |  |  | FLOF RATE THRU PIPING $\frac{.3208 \text { 天 GPM }}{\text { AREA }(\text { PIPE1. } 1 .)}=V$ |  |  |  | $\begin{array}{r} \text { D"SENES }=2.36 \\ \text { PEA I"OF GE } \end{array}$ |  |  |  |  |  |  |  |  |

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